
An Investigation into the Nonlinearity of Polyethylene Reactor Operation

M.-A. Benda, K. McAuley and P.J. McLellan
Department of Chemical Engineering
Queen's University

Outline

- Motivation
- Background: nonlinearity measures
- Scaling
- Introduce the polyethylene model
- Examples - Visual Interpretation of Curvature
- Measuring Curvature of the Polyethylene Model
- Controller Performance
- Summary
- Conclusions and Recommendations

Motivation

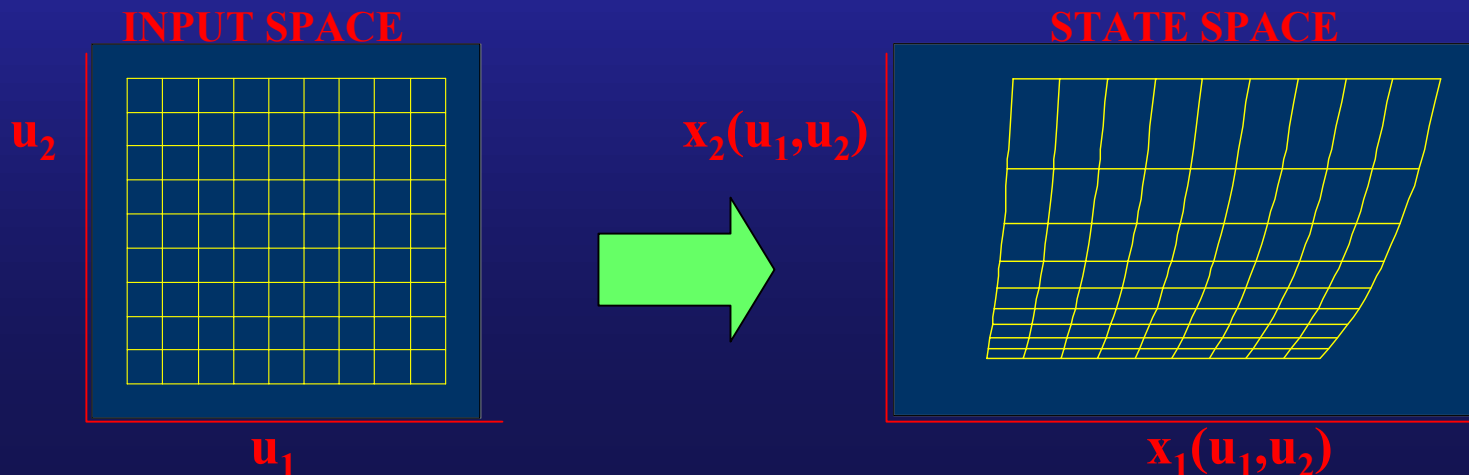
- inherent nonlinearity of chemical processes
 - bilinear -- bulk flow; exponential + ratios -- kinetics with adsorption; power law -- heat transfer ...
- linear control techniques more immediately accessible
 - mathematical framework and knowledge base in control practice
- *are there performance costs associated with relying on linear techniques?*
- **broad goal** - efficient allocation of control engineering resources
- a number of measures of nonlinearity have been proposed
- **specific goal** - examine nonlinearity of industrial process - gas-phase polyethylene reactor - product property control
 - fundamental model available - McAuley (1992)

Measures of Nonlinearity

- linearity properties - scaling - graphical test
- changes in linear approximations
 - Ogunnaike et al.
- deviations of linear approximations from full nonlinear behaviour
 - Allgower, Nikolaou
- curvature assessment using derivative information
 - Guay et al.
- **optimal control structure - closed-loop approach**
 - Stack and Doyle III

Nonlinearity and The Steady-State Locus

- The steady-state locus, $\Xi(\mathbf{u})$, is a hyper-surface that describes steady-state input/state behaviour
- $\Xi(\mathbf{u})$ is parameterized by P inputs and resides in N -dimensional state space
- Nonlinearity assessment is done by examining the local geometry of the steady-state locus - Guay et al. (1995)



Manifestation of Nonlinearity

There are two types of nonlinearity:

- 1) Parameter effects, or tangential acceleration
- 2) Intrinsic nonlinearity, or the normal component of nonlinearity

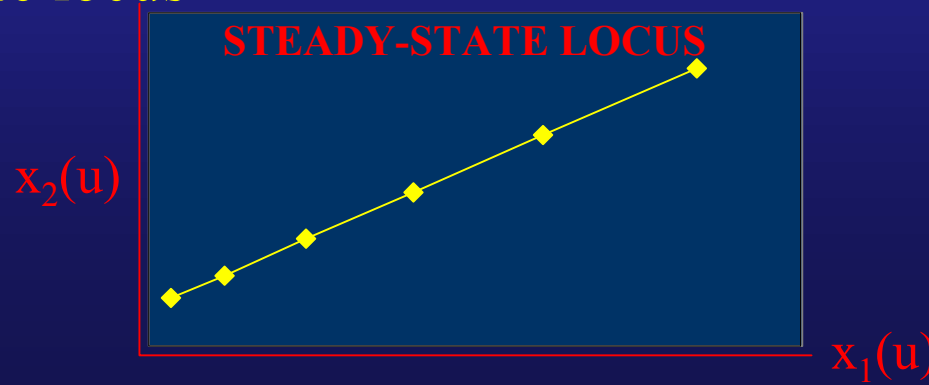
Manifestation of Nonlinearity

There are two types of nonlinearity:

- 1) Parameter effects, or tangential acceleration
- 2) Intrinsic nonlinearity, or normal component of nonlinearity

1) Tangential Acceleration

- Due to the change in *magnitude* of the model gains
- Presents itself as non-uniform spacing of points on the steady-state locus



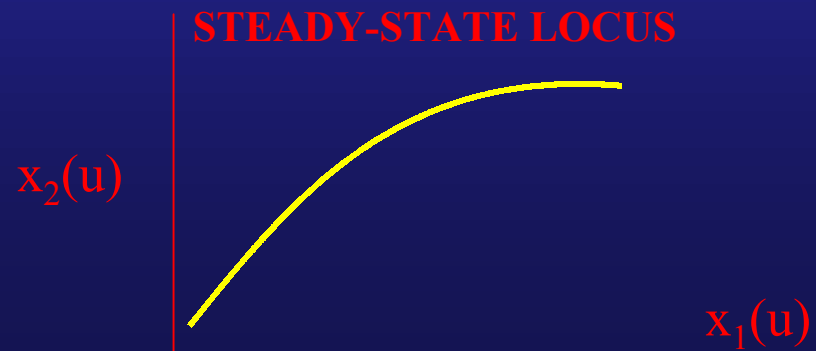
Manifestation of Nonlinearity

There are two types of nonlinearity:

- 1) Parameter effects, or tangential acceleration
- 2) Intrinsic nonlinearity, or normal component of nonlinearity

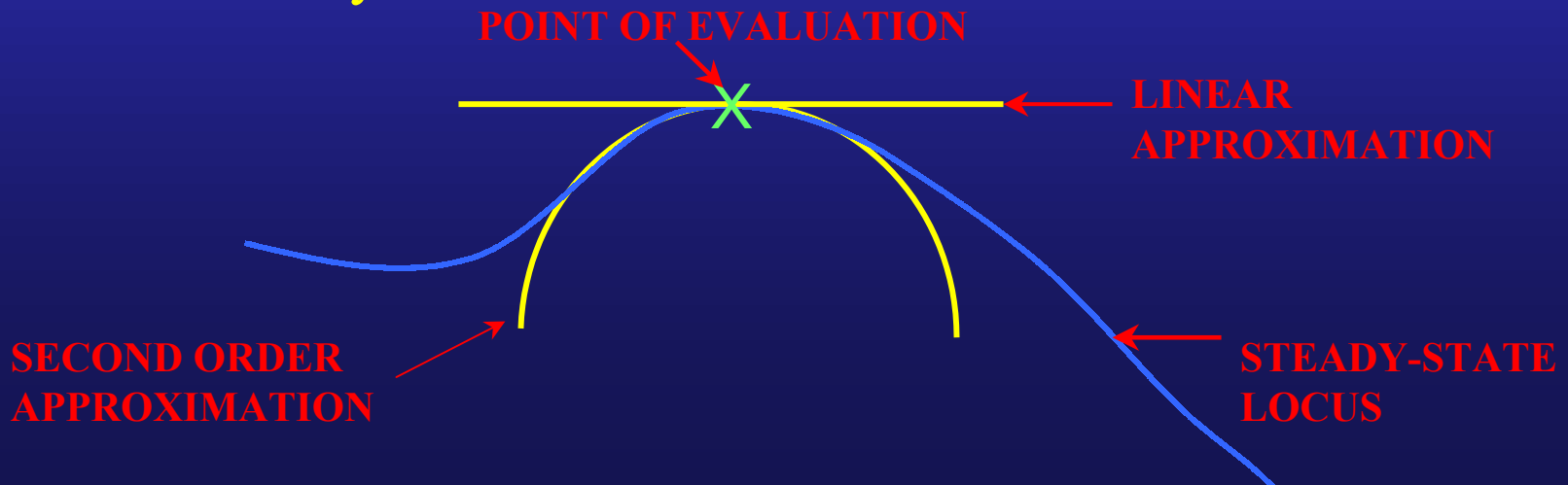
2) Normal Component of Nonlinearity

- Due to the change in *direction* of the model gains
- Presents itself as the curvature of the steady state locus relative to the surrounding state space.
- Present only if $P < N$



Assessing Nonlinearity

- A model is nonlinear if a nonzero second derivative with respect to inputs exists
- $\Xi(\mathbf{u})$ is locally approximated by first and second order Taylor series expansions
- The degree to which the first order approximation differs from the second order approximation gives an indication of nonlinearity.



Assessing Nonlinearity

- The degree of nonlinearity is evaluated by comparing the magnitude of the second order terms to the magnitude of the first order terms of the Taylor series approximation

- The first order terms (linear terms) are velocities, or gains:

$$\dot{\mathbf{v}}_i = \frac{\partial \mathbf{\Xi}(\mathbf{u})}{\partial u_i}, \quad 1 \leq i \leq P$$

- The second order terms (nonlinear terms) are accelerations:

$$\ddot{\mathbf{v}}_{pq} = \frac{\partial^2 \mathbf{\Xi}(\mathbf{u})}{\partial u_p \partial u_q}, \quad 1 \leq \{p, q\} \leq P$$

- The curvature, c_e , of $\mathbf{\Xi}(\mathbf{u})$ in a specific input direction, e is:

$$c_e = \frac{\|\text{acceleration}_e\|}{\|\text{velocity}_e\|^2}$$

A Measure of Average Nonlinearity

- c_e is a measure of curvature along a specific input direction, e

$$c_e = \frac{\|\text{acceleration}_e\|}{\|\text{velocity}_e\|^2}$$

- A more useful technique would be to average the curvature over the entire operating region
- Root Mean Square curvature (c_{RMS}) integrates curvature over all input directions.
- Guay, McLellan and Bacon (1995) suggested that significant nonlinearity is indicated by a $c_{\text{RMS}} > 0.3$

Scaling

- Scale to reflect intended operating region
- Scaling removes the unit dependence of RMS curvature on state variables
- RMS curvatures are based on a standard scaling which affords a generic measure of nonlinearity
- The intended region of operation is approximated by an ellipsoid

$$\Delta \mathbf{x}'(\mathbf{S}'\mathbf{S})\Delta \mathbf{x} = 1 \quad \text{where} \quad \Delta \mathbf{x} = \mathbf{x} - \mathbf{x}_0, \quad \mathbf{S} = \text{scaling matrix}$$

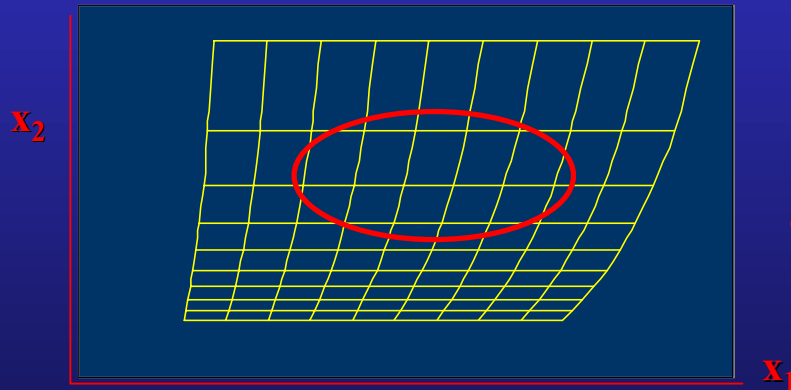
- Scale ellipsoidal region to unit spheroid (Guay et al., 1995)

$$\mathbf{z}'\mathbf{z} = 1 \quad \text{where} \quad \mathbf{z} = \mathbf{S}\Delta \mathbf{x}$$

Scaling

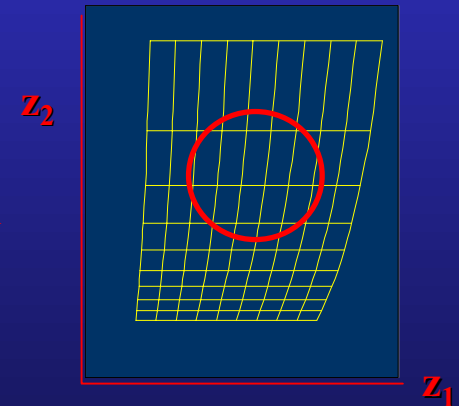
- RMS curvatures are calculated based on the unit sphere scaling

STEADY-STATE LOCUS

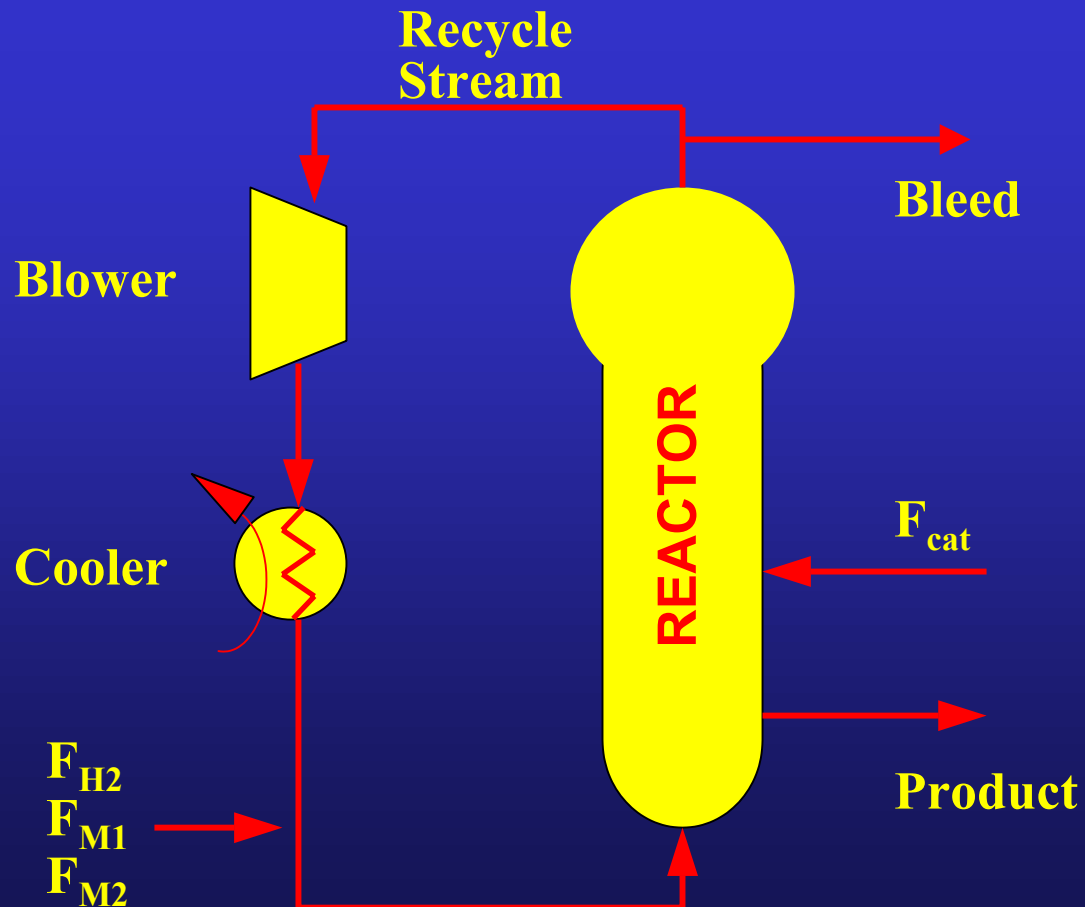


scaling

SCALED STEADY-STATE LOCUS



Polyethylene Reactor



Polyethylene

Gas Mass Balance Model

$$\frac{d[H_2]}{dt} = 0 = \frac{1}{Vg} \left(F_{H_2} - \frac{k_h \cdot F_{cat} \cdot a_{cat} \cdot [H_2]}{\frac{PR}{Bw} + k_d} - \frac{[H_2] \cdot b}{C_t} - gl \cdot [H_2] \right)$$

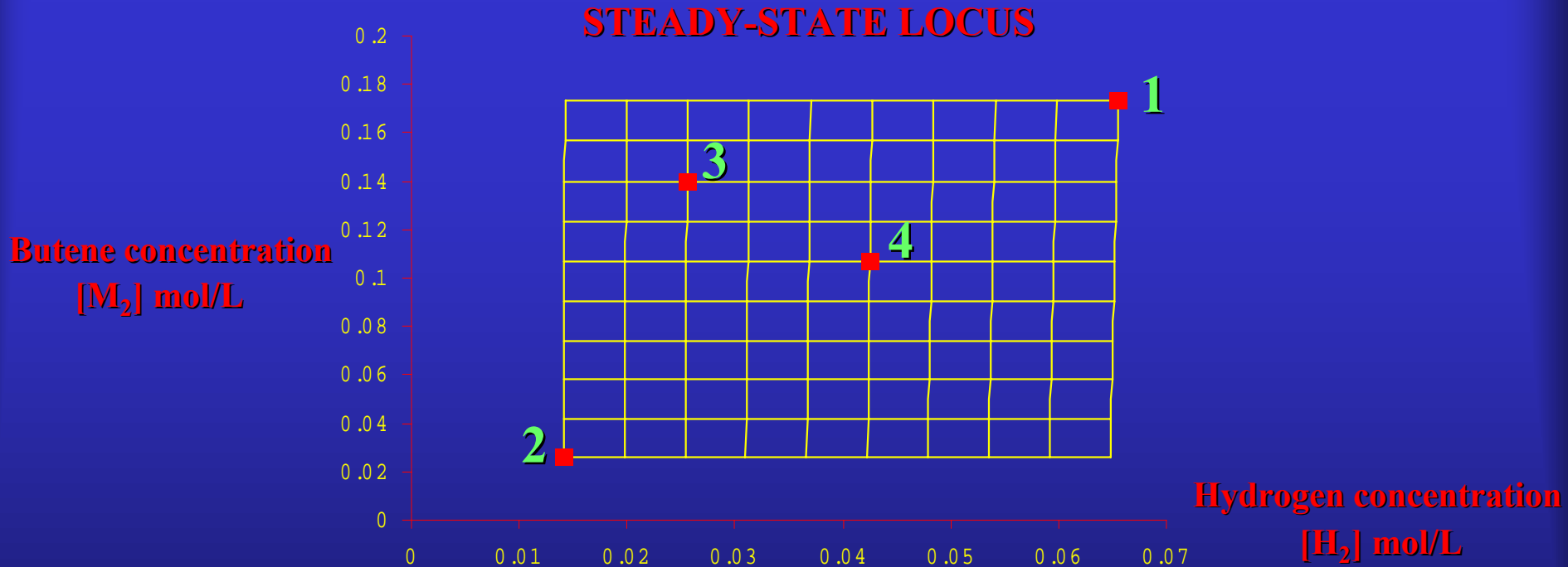
$$\frac{d[M_2]}{dt} = 0 = \frac{1}{Vg + Vs} \left(F_{M_2} - \frac{k_{p2} \cdot F_{cat} \cdot a_{cat} \cdot [M_2]}{\frac{PR}{Bw} + k_d} - \frac{[M_2]b}{C_t} - S[M_2]PR \right)$$

$$\frac{dBw}{dt} = 0 = F_{cat} a_{cat} \frac{(k_{p1}[M_1]mw_1 + k_{p2}[M_2]mw_2)}{\frac{PR}{Bw} + k_d} - PR$$

Assumptions:

- The total concentration C_t , and ethylene concentration $[M_1]$, are held constant by control
- The bleed flow is kept constant by a flow controller
- The reactor is operating under perfect bed weight control
- Temperature effects are negligible

Visualizing Curvature - Example 1



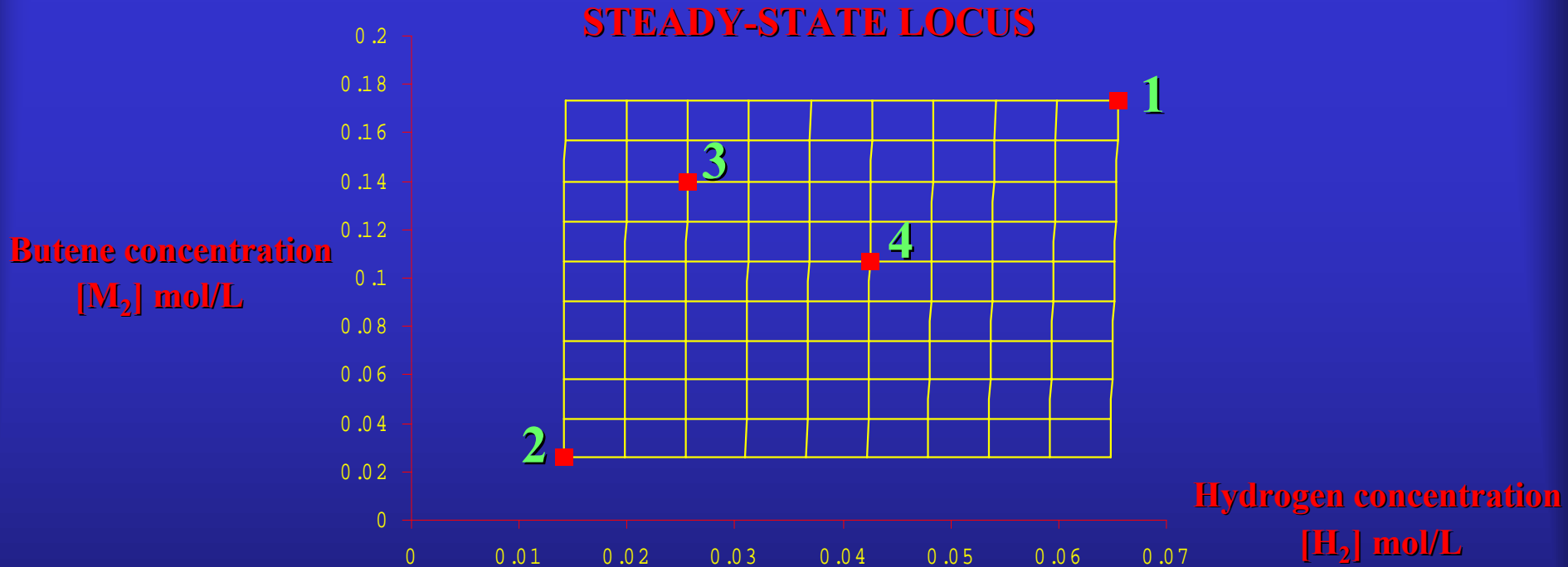
Input conditions:

$F_{\text{cat}} = 5.6 \text{ kg/h}$,
Fixed

$500 \leq F_{\text{H}_2} \leq 2300 \text{ mol/h}$,
Varying

$10000 \leq F_{\text{M}_2} \leq 64000 \text{ mol/h}$
Varying

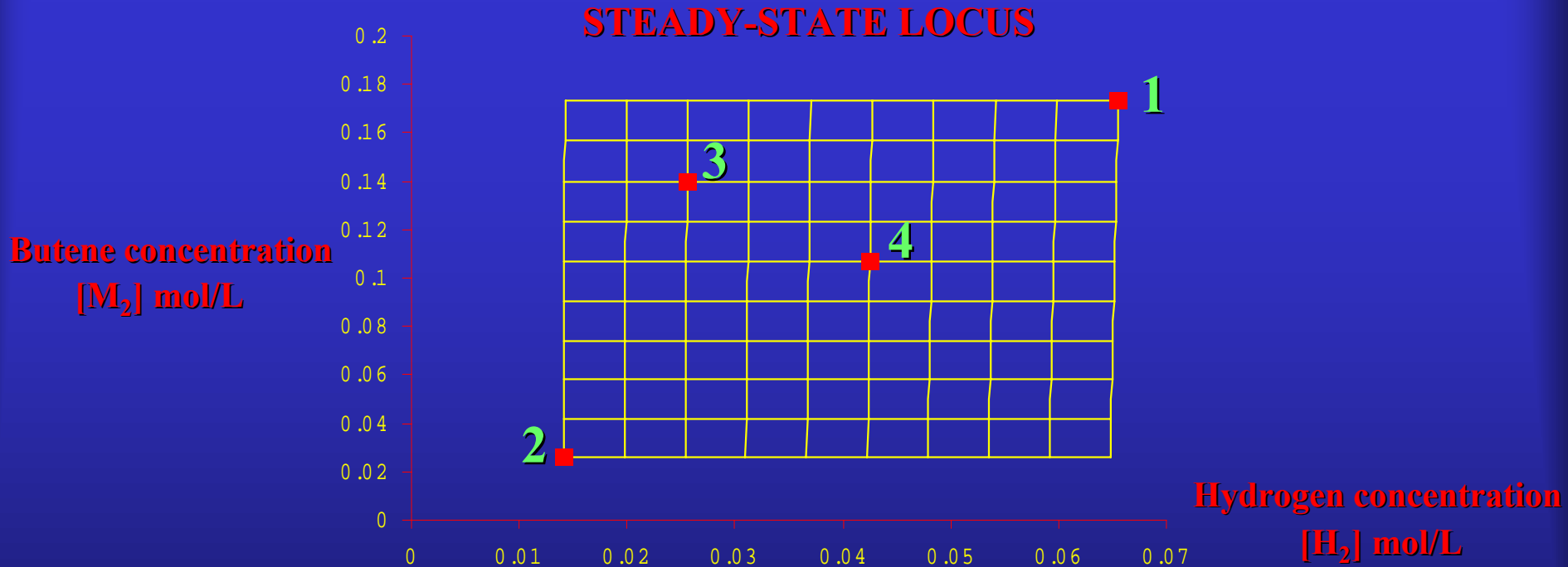
Visualizing Curvature - Example 1



Visual Inspection

- no curve to constant input lines
- equally spaced points
- implies little or no nonlinearity

Visualizing Curvature - Example 1

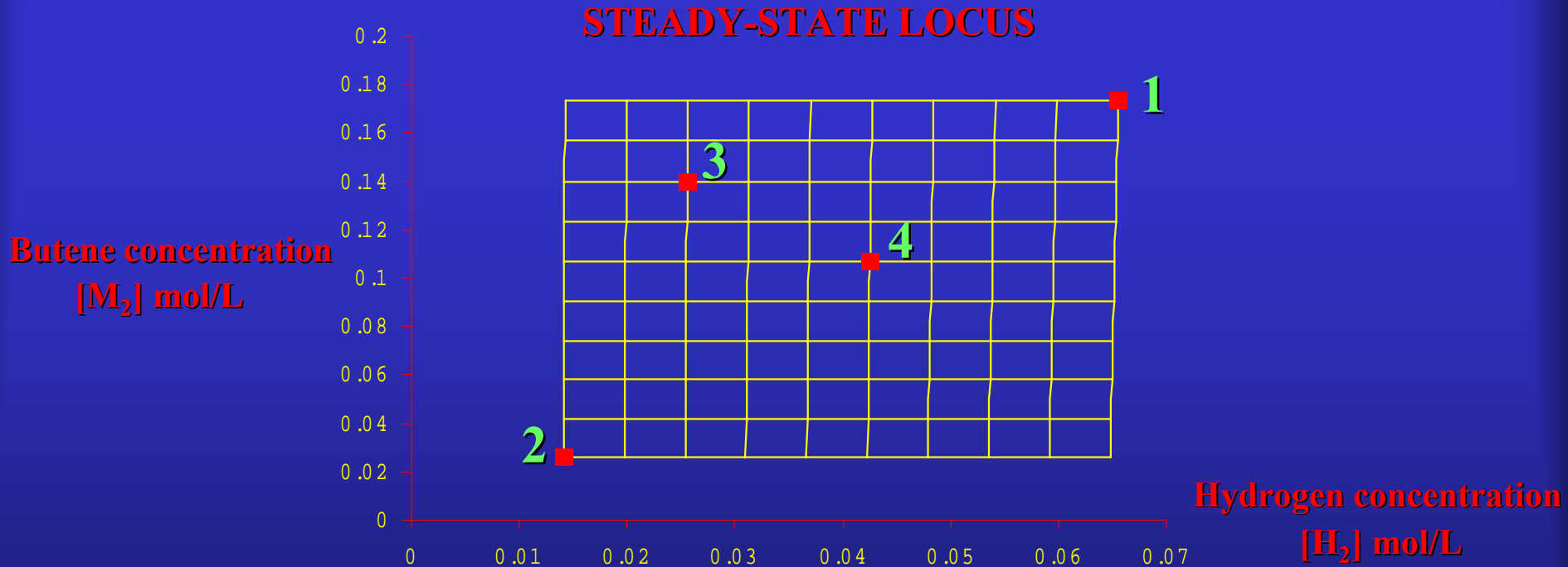


Visual Inspection

- no curve to constant input lines
- equally spaced points
- implies little or no nonlinearity

Point	RMS value
1	0.023
2	0.024
3	0.023
4	0.024

Visualizing Curvature - Example 1

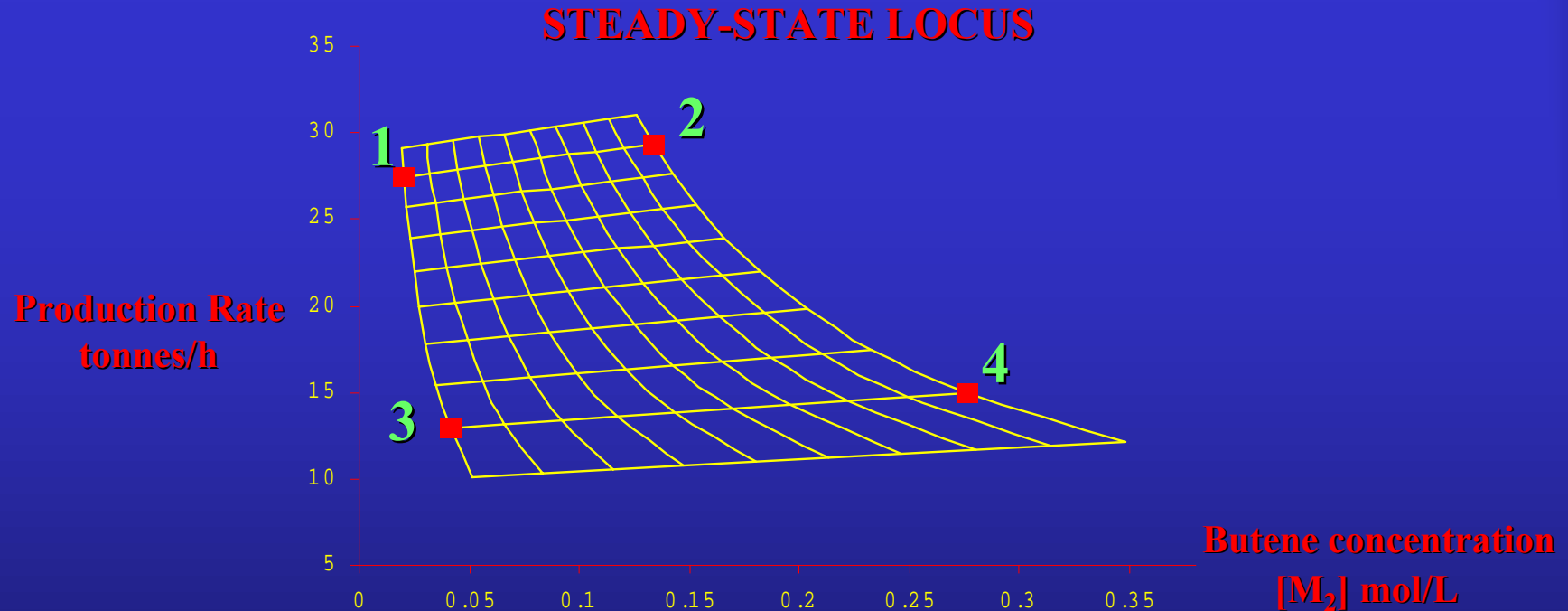


Control Expectation:

Control of this system should be accomplished well with a linear controller

Point	RMS value
1	0.023
2	0.024
3	0.023
4	0.024

Visualizing Curvature - Example 2



Input conditions:

$F_{H_2} = 1500 \text{ mol/h}$,

Fixed

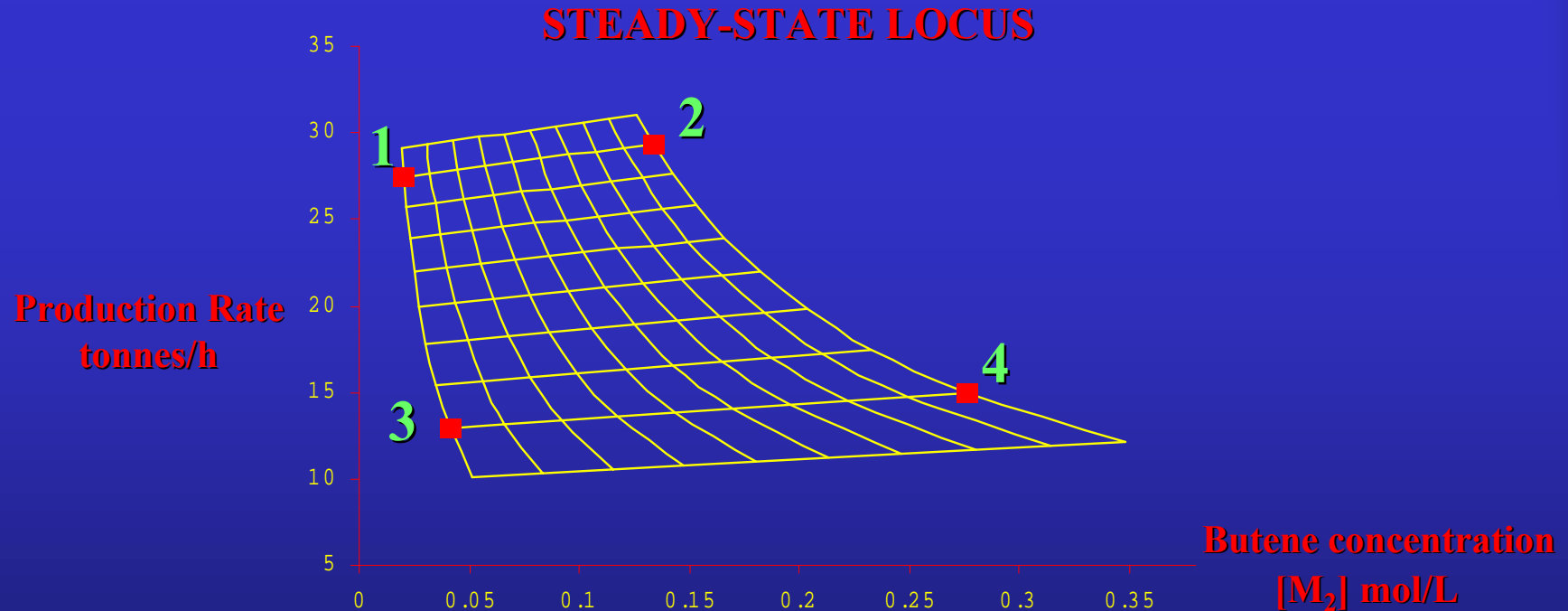
$2.0 \leq F_{cat} \leq 9.2 \text{ kg/h}$,

Varying

$10000 \leq F_{M_2} \leq 64000 \text{ mol/h}$

Varying

Visualizing Curvature - Example 2

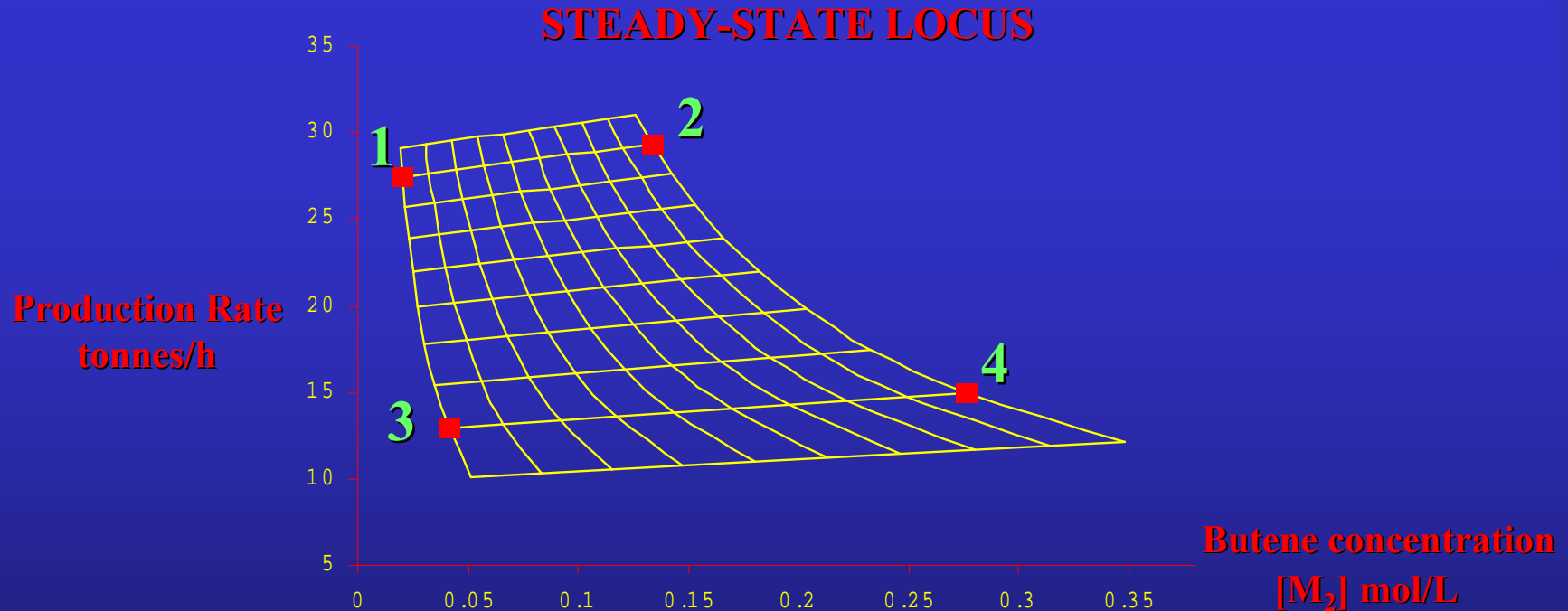


Visual inspection

- Point 1 shows the least curve in the constant input lines. Also most even spacing of lines

Point	RMS value
1	0.23

Visualizing Curvature - Example 2

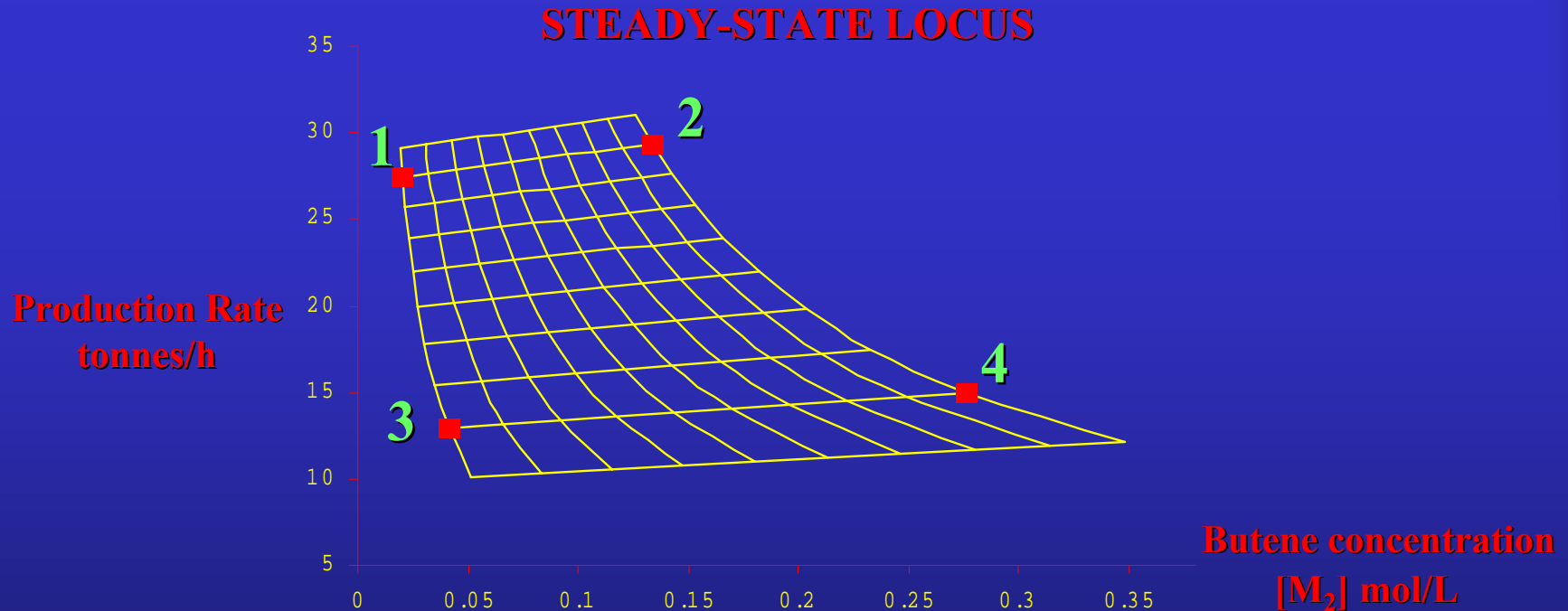


Visual inspection

- Point 1 shows the least curve in the constant input lines. Also most even spacing of lines.
- Point 2 has a bit more curve to the constant input line. Perhaps slightly more uneven spacing.

Point	RMS value
1	0.23
2	0.31

Visualizing Curvature - Example 2

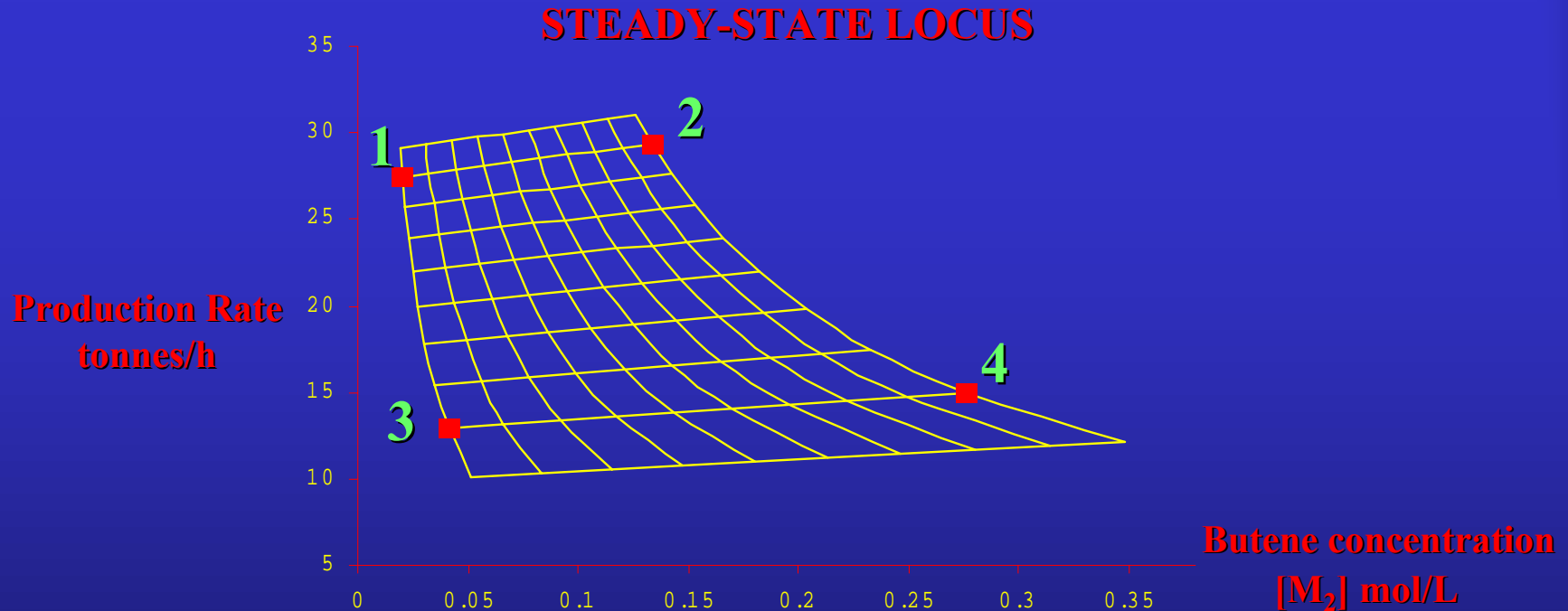


Visual inspection

- Point 3 shows more uneven spacing and more curve of constant input lines than at point 1

Point	RMS value
1	0.23
2	0.31
3	0.76

Visualizing Curvature - Example 2

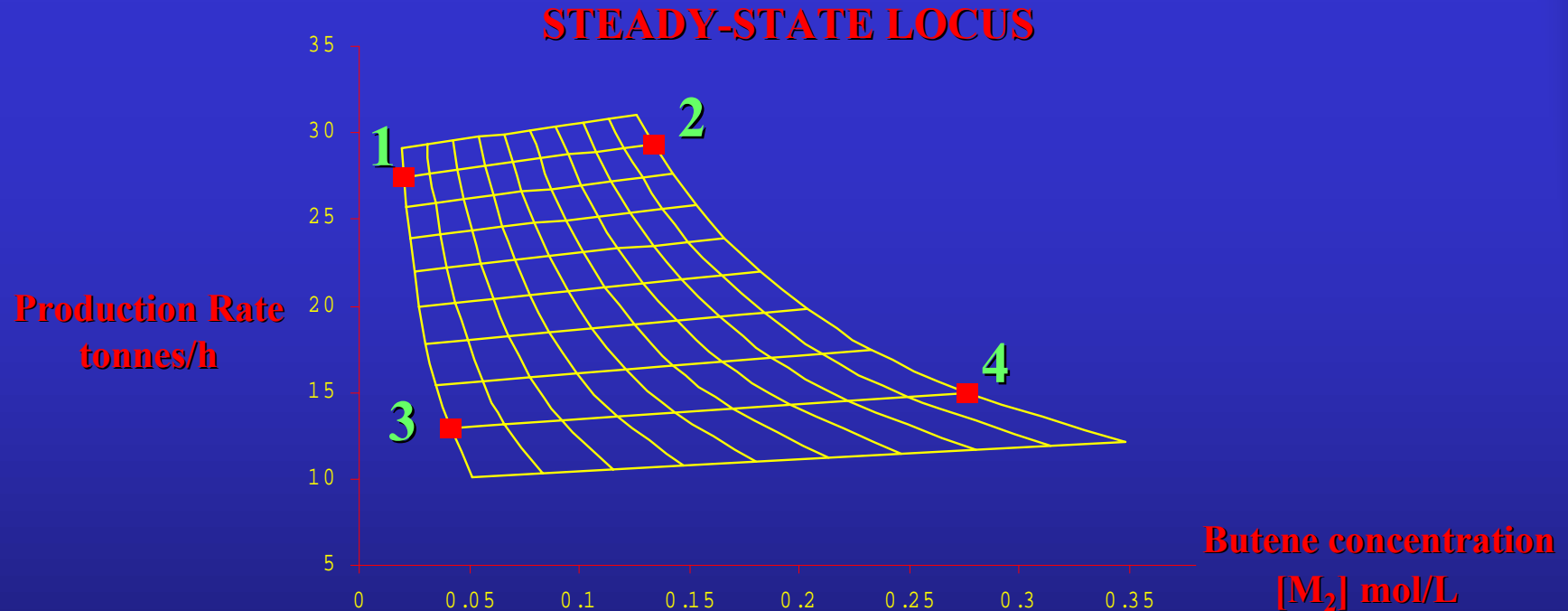


Visual inspection

- Point 3 shows more uneven spacing and more curve of constant input lines than at point 1
- Point 4 displays most curve and the most uneven spacing

Point	RMS value
1	0.23
2	0.31
3	0.76
4	1.90

Visualizing Curvature - Example 2



Control Expectations:

- If operating at point 1, linear control will suffice.
- If operating elsewhere on the locus, a nonlinear controller is highly recommended.

Point	RMS value
1	0.23
2	0.31
3	0.76
4	1.90

Curvature of the Full P.E. Model

- Polyethylene product property model
- The full PE reactor model consists of:
 - a dynamic gas mass balance of the form:

$$\dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}, \mathbf{u})$$

- a static output mapping of the form:

$$\mathbf{y} = \mathbf{h}(\mathbf{x})$$

- » melt index
- » density
- » production rate

The Full Polyethylene Reactor Model

$$\frac{d[H_2]}{dt} = \frac{1}{Vg} \left(F_{H_2} - k_H \cdot Y \cdot [H_2] - \frac{[H_2] \cdot b}{C_t} - gl \cdot [H_2] \right)$$

$$\frac{d[M_2]}{dt} = \frac{1}{Vg + Vs} \left(F_{M_2} - k_{p2} \cdot Y \cdot [M_2] - \frac{[M_2]b}{C_t} - S[M_2]Y(k_{p1}[M_1]mw_1 + k_{p2}[M_2]mw_2) \right)$$

$$\frac{dY}{dt} = F_{cat} \cdot a_{cat} - \frac{Y^2(k_{p1}[M_1]mw_1 + k_{p2}[M_2]mw_2)}{Bw} - k_d Y$$

$$\ln(MI) = 3.5 \ln \left(k_0 + k_1 \frac{[M_2]}{[M_1]} + k_3 \frac{[H_2]}{[M_1]} \right)$$

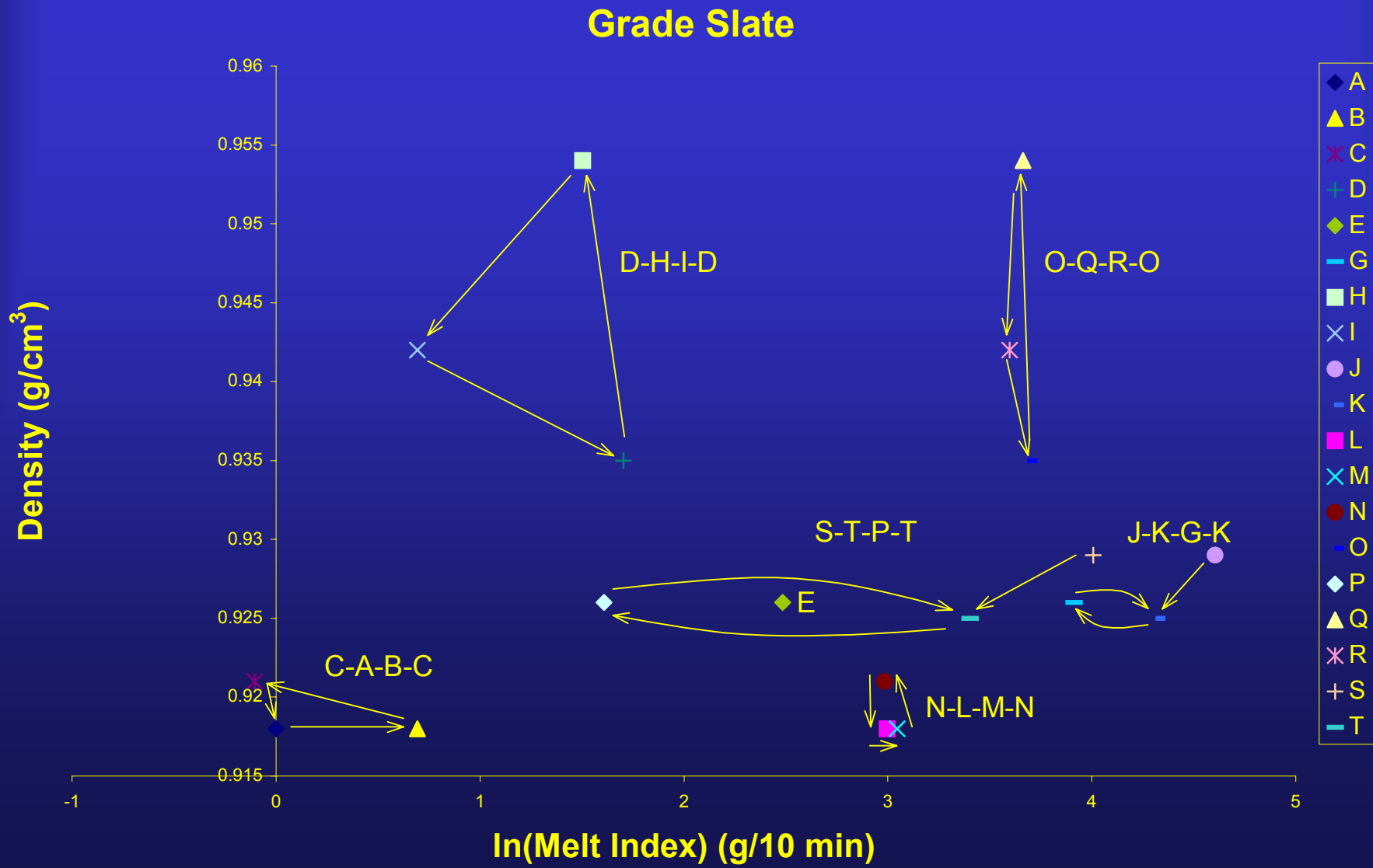
$$\rho = p_0 + p_1 \ln(MI) - \left(p_2 \frac{[M_2]}{[M_1]} \right)^{p_4}$$

$$PR = Y(k_{p1}[M_1]mw_1 + k_{p2}[M_2]mw_2)$$

Nonlinearity & Performance

- Is steady-state nonlinearity a good indicator of closed-loop performance degradation under linear control?
- Approach
 - design linear and nonlinear controllers
 - » linear output feedback - pole placement on tracking error trajectory
 - » input-output linearizing controller - pole placement on tracking error trajectory
 - simulate under setpoint tracking and disturbance rejection scenarios
 - compare relative performance

Polyethylene Grade Selection



Disturbances for Control Simulations

- 25% increase in a_{cat} (concentration of active sites)
 - due to batch to batch variation
- 25% increase in k_d (deactivation of active sites)
 - due to reactive impurities
- 25% decrease in b (bleed flow)
 - due to operator intervention or pressure changes downstream of the bleed valve

Melt Index Transformation

- Most practitioners use the following model form, because the logarithm transform is thought to provide better linear control:

$$\mathbf{y}(\mathbf{x}) = \begin{bmatrix} \ln(MI) \\ \rho \\ PR \end{bmatrix}$$

- As opposed to the nominal model (non-transformed model):

$$\mathbf{y}(\mathbf{x}) = \begin{bmatrix} MI \\ \rho \\ PR \end{bmatrix}$$

Calculating RMS Curvature

- Note the large difference in RMS curvature between the transformed model and the nominal model:

Grade Transition	Curvature Transformed Model	Curvature Nominal Model
S -T-P-T	0.813	117
D-H-I-D	0.951	14.0
C-A-B-C	1.70	6.70
J-K-G-K	2.77	89.6
O-Q-R-O	18.6	20.5
N-L-M-N	70.2	72.4

Linking Nonlinearity to Controller Performance

- have curvature of many regions on the output space
- use this information in comparative controller performance study
- determine if consistent relationship between curvature and linear control performance is observed

Performance Measure

- Designed an error measure that
 - Compares linear to nonlinear control performance
 - Is a generic normalized measure
 - Is based on IAE
- For grade transition simulations, compared the design trajectory to the actual trajectory
 - IAD_{norm}
- For regulatory control simulations, compared the actual output against the set point.
 - IAE_{norm}

Performance Measure

- IAD: Integral of the deviation between designed and actual trajectory
- IAE: Integral of the error between set point and actual trajectory
- Difference between deviations/errors of the linear and nonlinear controllers:
 - $IAD = IAD_{\text{linear}} - IAD_{\text{nonlinear}}$
 - $IAE = IAE_{\text{linear}} - IAE_{\text{nonlinear}}$

Performance Measure

- Normalize the performance measure with a nominal deviation/error value
 - For grade transitions:

$$IAD_{\text{norm}} = \frac{IAD_{\text{linear}} - IAD_{\text{nonlinear}}}{IAE_{\text{nominal}}}$$

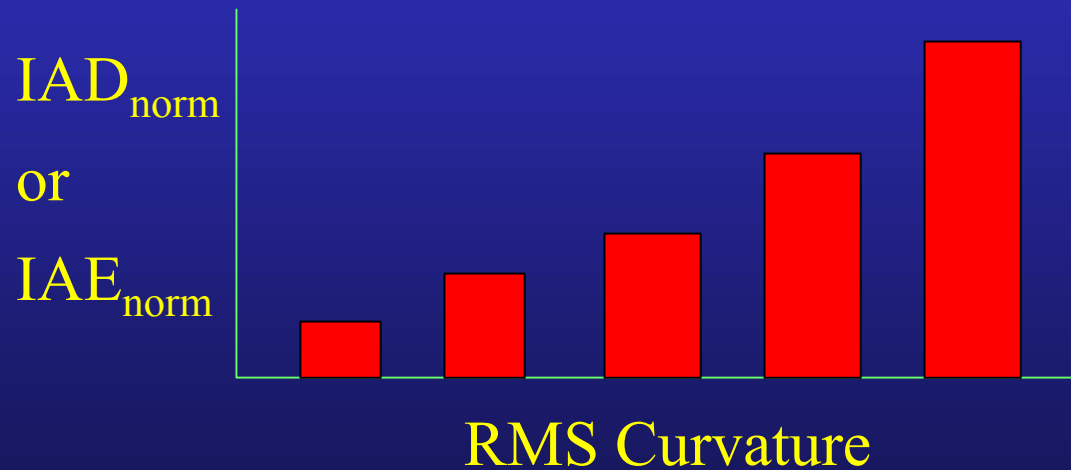
- For disturbance rejection:

$$IAE_{\text{norm}} = \frac{IAE_{\text{linear}} - IAE_{\text{nonlinear}}}{IAE_{\text{nominal}}}$$

- where $IAE_{\text{nominal}} = \int |(SP - CV_{\text{design}})| dt$

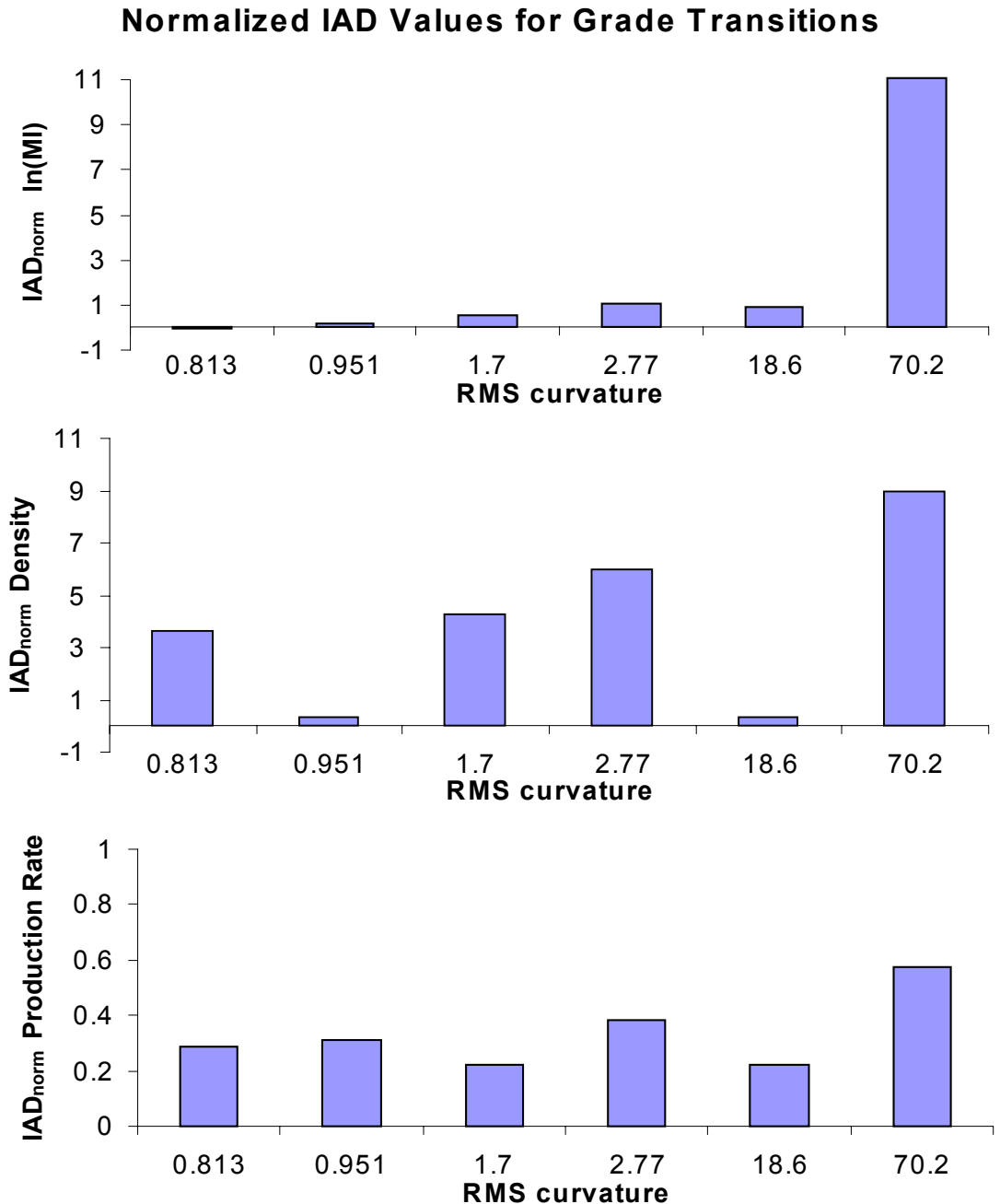
Nonlinearity and Controller Performance

- Expected Relationship:



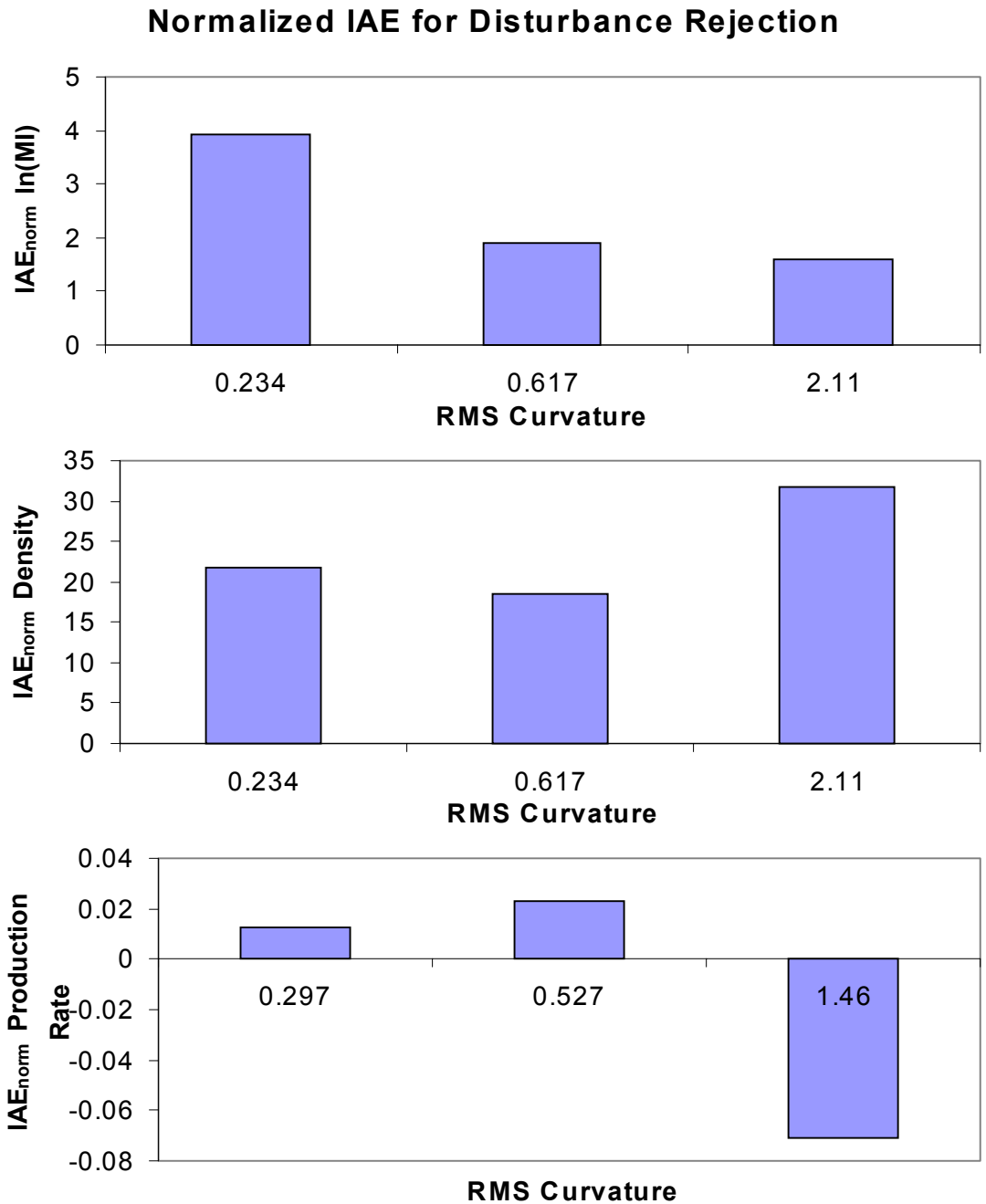
Results

- Grade Transition Simulations



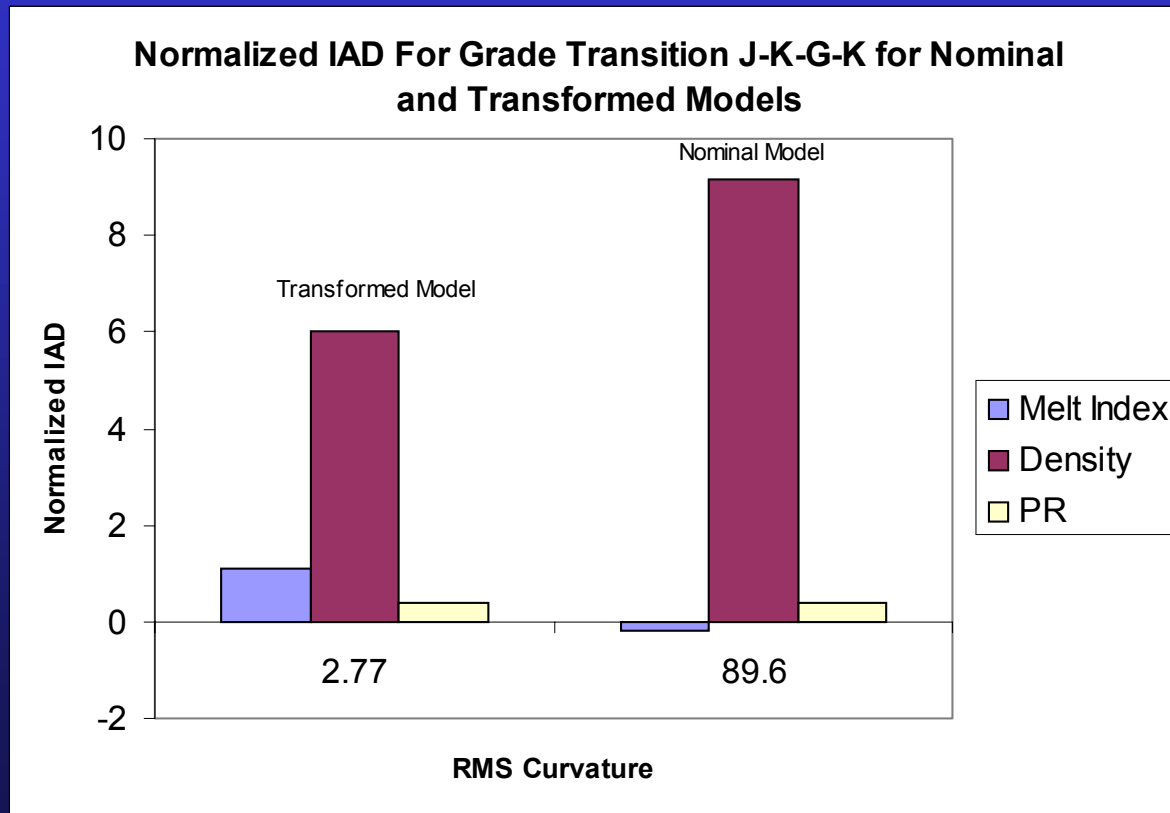
Results

- Disturbance Rejection Simulations



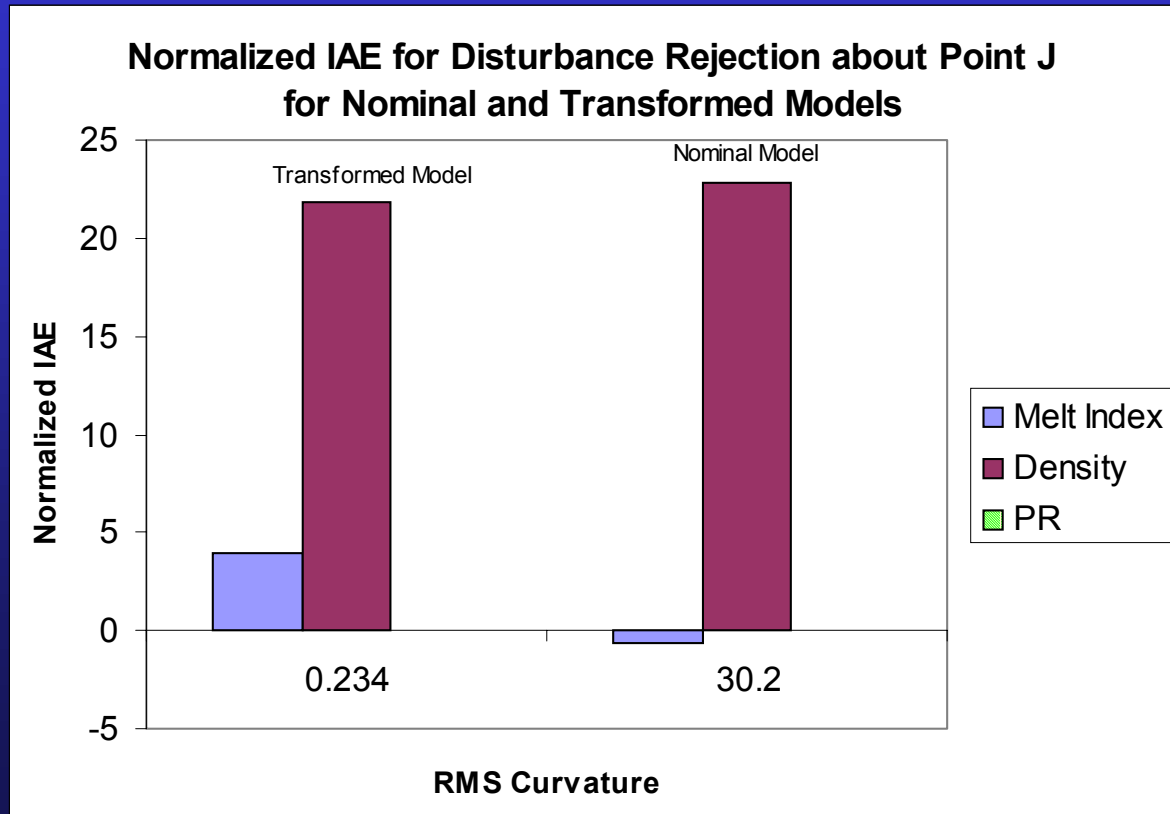
Results

- Grade Transition J-K-G-K
- Transformed vs. Nominal Model



Results

- Regulatory control about point J
- Transformed vs. Nominal Model



Result

- There is no consistent relationship between steady-state curvatures and performance for this example.
- Possible reasons
 - Manipulated Variable Saturation - investigated - no
 - Performance Measure Inappropriate - relative measure defined - consider L_2 measure?
 - Directionality Effects - investigated for specific directions
 - Dynamic vs. Steady-State Curvature?
 - Comparison Measure - accounting for MV action?

Summary

- Nonlinearity measures
- Scaling issues
- Polyethylene model
- Steady-state curvatures
- Curvature and performance depending on operating position and mode
 - grade transitions - “servo problem”
 - disturbance rejection
 - linear state feedback
 - input-output linearization + pole placement

Conclusions

- Mild to moderate nonlinearity was detected in the state (gas mass balance) model
- Mild to severe nonlinearity was detected in the output (full PE) model
- The nominal model (melt index) displays a higher degree of curvature than the transformed model ($\ln(\text{MI})$)
- There is no consistent relationship between steady-state curvature and controller performance for this process example.
- Nonlinearity does not always imply degradation at a point
 - approximation error can produce output response with reduced performance cost
 - need to penalize MV input action

Acknowledgement

The financial support NSERC is gratefully acknowledged.